

Industrial Systems and Control Limited

*“Control Engineering Services for
the Automotive Industry”*

ISC Background

- Control engineering consultancy
 - Founded 1987 - as a spin-out from Strathclyde University
- Works across many sectors:
 - Automotive; Marine; Oil/Gas; Power Generation
 - Small – 5 full time, 4 part time employees
- However we work with some very large companies:
 - General Motors; Toyota; Ford; Jaguar; Visteon; BP; Shell;
Boeing; BAE Systems; Rolls Royce Marine; SSE; Alstom; EDF

ISC Core Capabilities

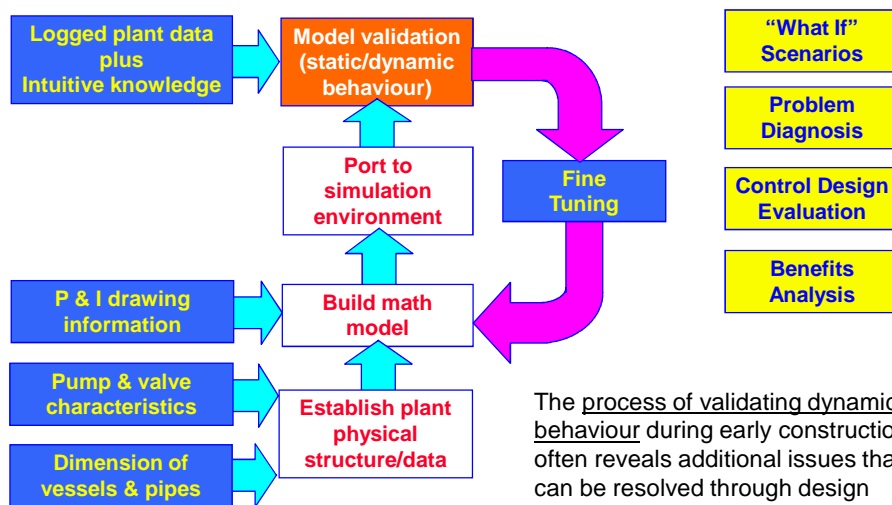


- Dynamic Systems Modelling:
 - high fidelity first principles models or data-driven identification
- Control Strategy Design – both new and improvements
 - full software development – NI LabVIEW / CompactRio
- Troubleshooting Control Problems
- Optimisation
- Energy Efficiency Benefits Analysis
- Technology Reviews
- Training – standard and bespoke courses

ISC provides support on novel or challenging control applications.

Close working with clients to get best solution and ensure full transfer.

Dynamic Modelling at the Core



ISC capabilities in Automotive



- Projects on Advanced Control:
 - General Motors, US – Nonlinear Control for Powertrain
 - GM – Estimation of Aging in Catalyst
 - GM – New Techniques for Engine Calibration
 - GM – Advanced Predictive Control for Improved Emissions/Fuel
 - Toyota, US – New Advanced Control for Automotive Powertrain
 - Torotrak - Variable transmission system
- Training: Chrysler, Ford, Freescale, GM, Jaguar, Ricardo, Toyota and Visteon.

ISC capabilities in Automotive



- ISC has National Instruments (NI) Powertrain Control Software and Hardware
 - complete set of actuator drivers and sensor IO
- Allows rapid control prototyping on test engines
- Customisable software using NI LabVIEW
 - Fast and flexible implementation
 - Replication and monitoring of ECU core functions



ISCs Control Engineering Training



Wide range of highly regarded industrial courses

From basic loop tuning to advanced control

Extensive computer-based hands-on



- When given at company premises can be tailored to exact need
- Bespoke exercises and entire courses can be provided
- Scheduled Training Courses
 - our five most popular and general courses
 - held in central location, scheduled regularly

ISC courses for Automotive Control

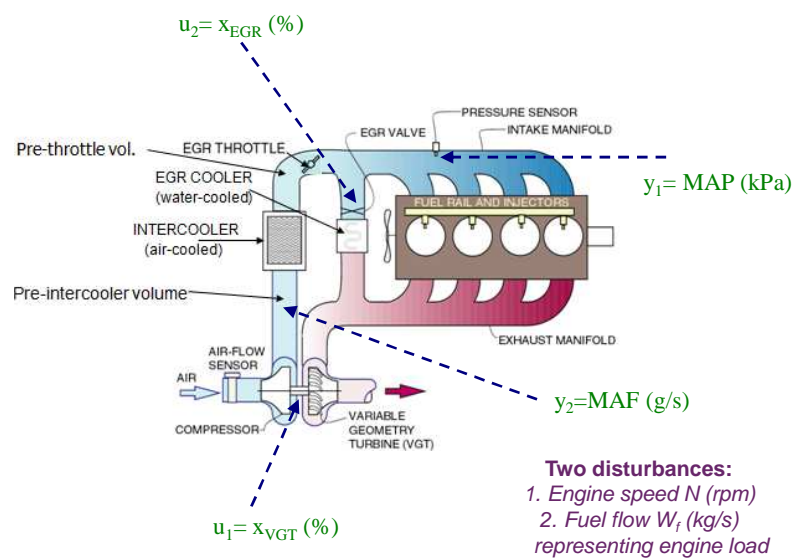


- Control Engineering Practice
- Control Fundamentals – System Identification
- Model Based Control for Automotive Applications
- Nonlinear Control for Automotive Applications
- Kalman Filtering
- Optimization and System Identification
- Introduction to Modelling, Simulation & Control for Automotive Applications

Automotive Control Projects

#1 Diesel Engine Control

Diesel Engine 2x2 System



Diesel Engine Model Example



- To control:
 - Intake manifold pressure or Manifold Absolute Pressure (MAP)
 - Exhaust gas recirculation (EGR) rate (or fraction)
- Two control signals:
 - Variable Geometry Turbine (VGT) vane lift u_{vgt} [% closed]
 - EGR valve stem position u_{egr} [% open]
- Includes additional actuator:
 - EGR throttle u_{thr} [% closed]
 - Normally kept fully open - can be used to extend authority of EGR valve

Diesel Engine Model Example



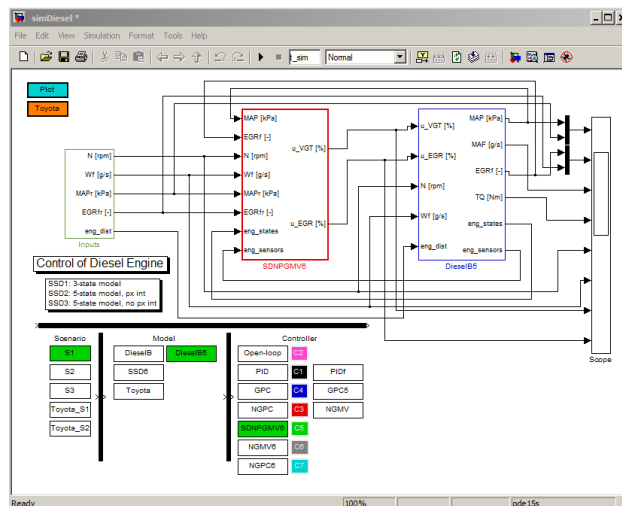
- Both VGT and EGR introduce internal feedback paths in engine
 - Potentially, depending on engine operating conditions, cause:
 - Response overshoots
 - Non-minimum phase behavior
 - DC sign reversal

Simulation Test Bed

- Allows selection of desired:

- Engine model
- Controller
- Simulation scenario

- Used in conjunction with MATLAB design script



Control Objectives

- Diesel engine control objectives
 - Supply driver's requested engine torque
 - Minimize fuel consumption
 - Keep NO_x conc. in exhaust and smoke as low as possible
- In this study, torque control not considered
 - Fuel flow W_f is given - represents engine load.
- EGR is ratio of exhaust gas flow recirculated into intake manifold to total gas flow into cylinders

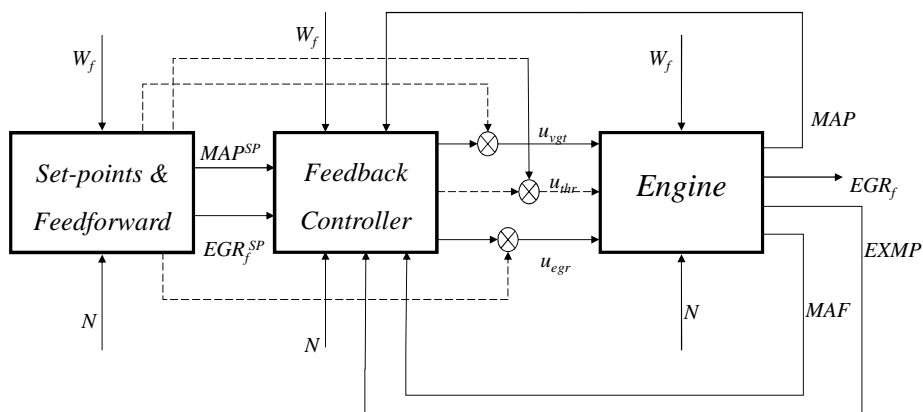
- Defined as:

$$EGR_f = \frac{W_{egr}}{MAF + W_{egr}}$$

Control Objectives

- Set-points for EGR_f and MAP are computed on-line from static maps (given as look-up tables)
 - Maps are functions of engine conditions - i.e. engine speed and fuel flow rate
- General block diagram of Diesel Engine control system may be constructed as in following Figure

Diesel Engine Control System Schematic Diagram

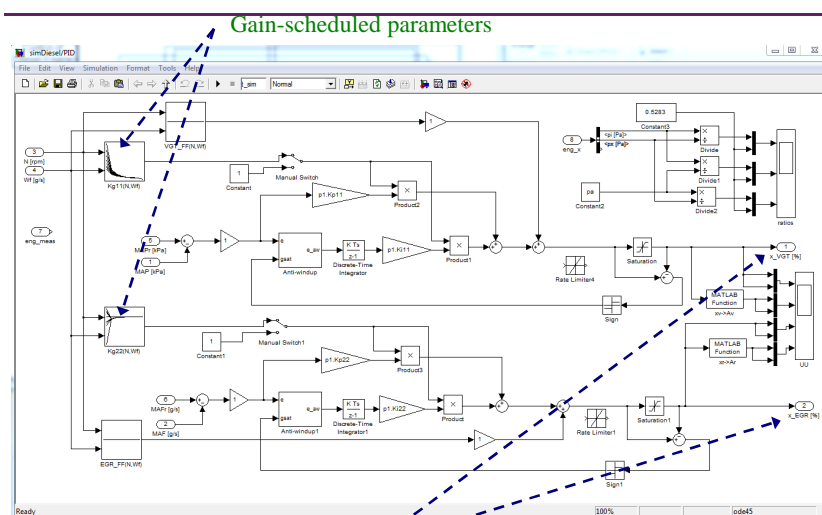


Control Structure



- EGR throttle assumed to remain fully open
 - To simplify control problem
 - Results in square 2×2 system
 - Feedforward signals not included explicitly in solution
 - Denoted by dashed lines in figure
- Feedback Controller block includes Kalman Filter
 - Estimates engine states based on:
 - Model
 - Measurements of MAP, MAF and exhaust manifold pressure EXMP

Gain-Scheduled PID Control



(x, x_r) used as controller outputs

Automotive Control Projects

#2 Predictive Control for SI and Diesel Engines

MPC Performance Index and Design

- Control vector $\Delta \mathbf{U}(t)$ is often costed and calculated to minimize:
 - Predicted errors over prediction horizon N
 - Size of control move over control horizon N_u
- Consider the quadratic performance index:

$$\min_{\Delta \mathbf{U}(t)} \mathbf{J} = \hat{\mathbf{E}}(t+1)^T \mathbf{Q} \hat{\mathbf{E}}(t+1) + \Delta \mathbf{U}(t)^T \mathbf{R} \Delta \mathbf{U}(t)$$

- where \mathbf{Q} is positive-definite weighting matrix and \mathbf{R} is positive semi-definite matrix
- Both \mathbf{Q} and \mathbf{R} are usually diagonal matrices with positive diagonal elements
- Weighting matrices are used to weight most important outputs and inputs

Design Guidance in using Predictive Control for Engines



- Methods to deal with uncertainties, disturbances and robustness
 - Cost weightings and additional sensitivity cost terms
 - Disturbance models
 - System descriptions
- Tuning Procedures for Robustness:
 - PID inspired / Free cost weighting choice
- Modifications to the Estimator:
 - Modelling error compensation
- Application of constraints:
 - Hard constraints / Soft constraints
- Delay compensation - similar ideal response as Smith Predictor

Integral Action



- Note steady state error
 - MPC (in basic form) has no integral action
 - Needs to be introduced artificially
- Recommended:
 - Introduce integrator on dynamic control error weighting
 - Weight incremental control action and modify plant by augmenting an integrator

State Dependent LPV Systems



- System with state equation matrices that depend upon:

- States
- Control inputs
- Parameters

$$x(t+1) = \mathcal{A}(x, u, p)x(t) + \mathcal{B}(x, u, p)u(t) + \mathcal{D}(x, u, p)d(t)$$
$$y(t) = \mathcal{C}(x, u, p)x(t) + \mathcal{E}(x, u, p)u(t)$$

State Dependent LPV Systems



- State-dependent LPV systems include:
 - Pseudo-LPV systems
 - PWA systems
 - Some Hybrid systems
- Choice of model is a problem:
 - Uniqueness
 - Type of model due to origin (physical equations, Jacobian, ...)
 - Uncertainty representation (unstructured, parametric, ...)
 - Numerical computations and sensitivity
 - Implications for Control law solution (bias terms, scaling, ...)

Cost-Function Definition: Engines



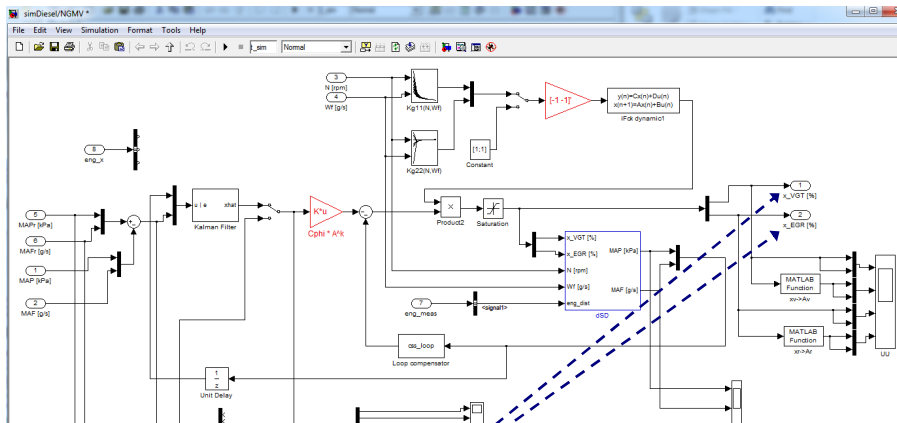
- Optimal control problem and cost-function must have simple base level
 - Which is well understood
 - But must also be able to cope with:
 - Future enhancements
 - Changes to performance, drivability & fatigue knowledge
- Generalized Predictive Control (GPC) related cost-function and solution approach was first proposed
 - Reason: very successful in process industries

Cost-Functions: GPC Augmentation



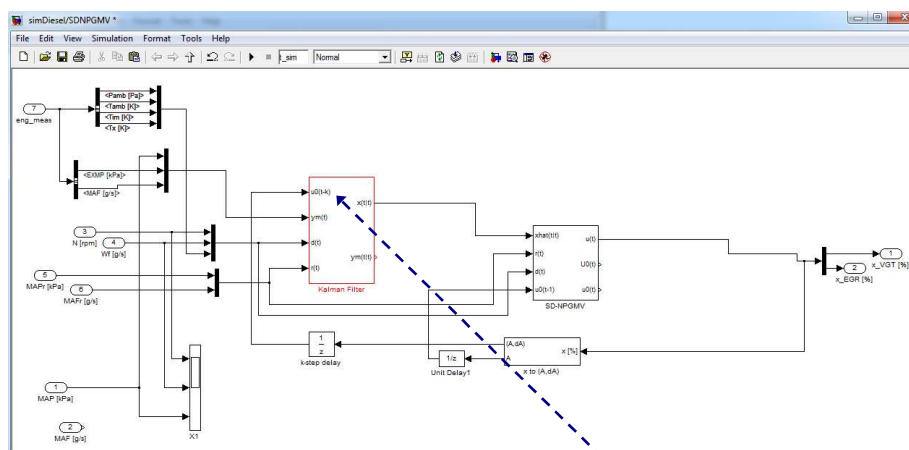
- Cost-function can however include dynamic weightings
 - To enable frequency response shaping of designs
- Weightings may approximate nonlinearities if made state-dependent
 - Possibly needed in drivability/fatigue minimization problem
 - Involves little additional complexity
- Cost-function will also include:
 - Traditional tracking error term; control action costing term
- Cn also include nonlinear control weighting term to introduce:
 - Wind-up protection; Hard constraint limits

NGMV Control



(x_v, x_r) used as controller outputs

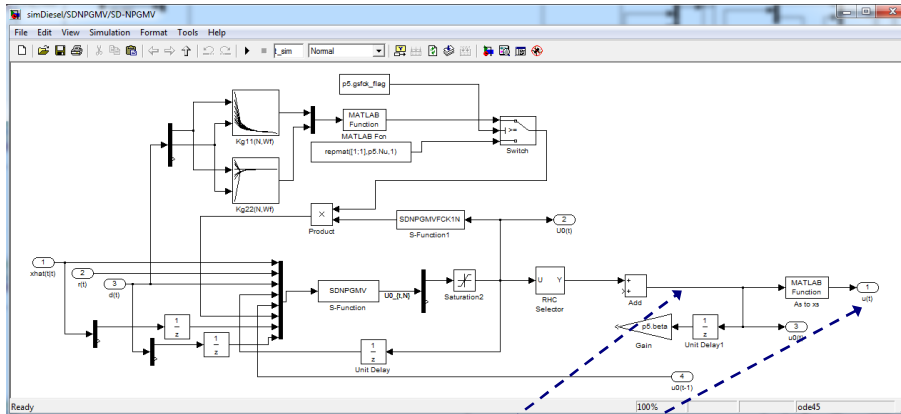
SD-NPGMV Controller Structure



$u(t)$ refers to (x_v, x_r) controls
 $u_0(t)$ refers to (A_v, A_r) controls

SD model used in Kalman Filter
 so $u_0(t-k)$ fed back

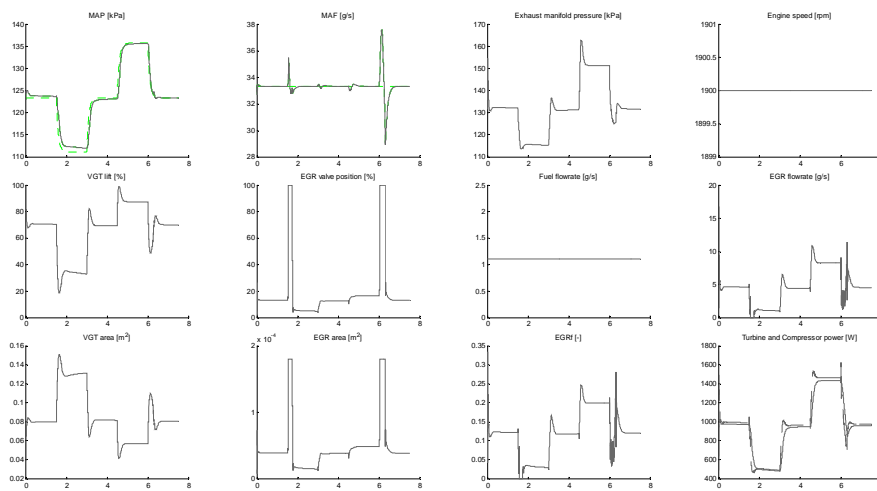
SD-NPGMV block



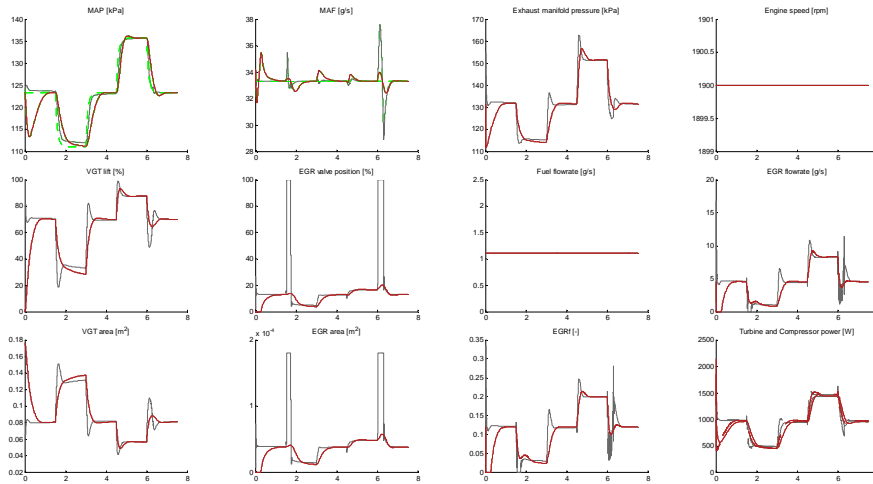
Controller computes optimal (A_v, A_r)

Transformed to (x_v, x_r)

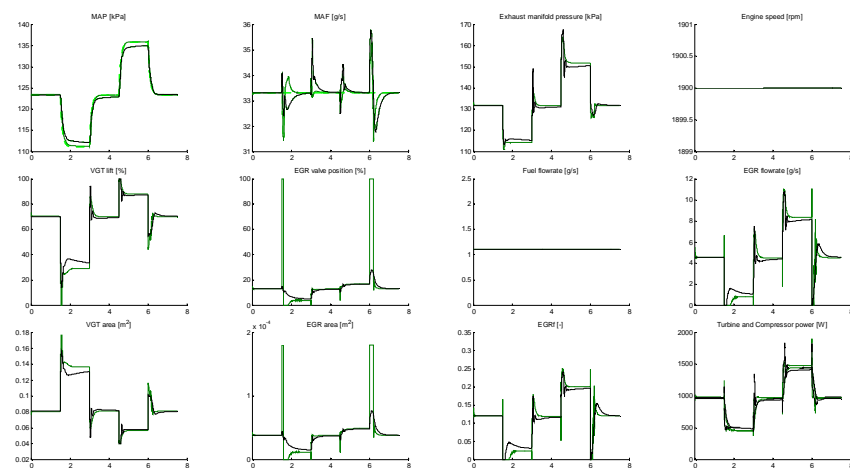
NGPC with PID Inspired Weightings and Dynamic Control Costing



Comparison of NGPC with Dynamic Control Costing and NGMV (red) both with PID Inspired Weightings



Comparison of NPGMV with PID Inspired Weightings and F_{ck} Costing and PID Controller (black)



Thank you. Any questions ?

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